Analysis of Statically Indeterminate Structures Using the Force Method Steven Vukazich San Jose State University

Statically Indeterminate Structures

At the beginning of the course, we learned that a **stable structure** that contains **more unknowns than independent equations of equilibrium** is **Statically Indeterminate**.

Advantages

- Redundancy (several members must fail for the structure to become unstable);
- Often maximum stresses is certain members are reduced;
- Usually deflections are reduced.

Disadvantages

- Connections are often more expensive;
- Finding forces and deflections using hand analysis is much more complicated.





Consider the beam



Define Primary Structure and Redundants

- Remove all applied loads from the actual structure;
- Remove support reactions or internal forces to define a primary structure;
- Removed reactions or internal forces are called redundants;
- Same number of redundants as degree of indeterminacy
- Primary structure must be stable and statically determinate;
- Primary structure is not unique there are several choices.



Define and Solve the Primary Problem

- Apply all loads on actual structure to the primary structure;
- Define a reference coordinate system;
- Calculate relevant deflections at points where redundants were removed.



Define and Solve the Redundant Problems

- There are the same number of redundant problems as degrees of indeterminacy;
- Define a reference coordinate system;
- Apply only one redundant to the primary structure;
- Write the redundant deflection in terms of the flexibility coefficient and the redundant for each redundant problem.
- Calculate the flexibility coefficient associated with the relevant deflections for each redundant problem;



Define and Solve the Redundant Problems



Compatibility Equations

Compatibility at Point C

$$\Delta_C + \Delta_{CC} + \Delta_{CD} = 0$$

Compatibility at Point D

$$\Delta_D + \Delta_{DC} + \Delta_{DD} = 0$$

Compatibility Equations in terms of Redundants and Flexibility Coefficients

$$\Delta_{C} + C_{y}\delta_{CC} + D_{y}\delta_{CD} = 0$$
$$\Delta_{D} + C_{y}\delta_{DC} + D_{y}\delta_{DD} = 0$$

Solve for C_v and D_v



Example Problem



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Compatibility Equation at Point A

Compatibility at Point A

$$\theta_A + \theta_{AA} = 0$$

Compatibility Equation in terms of Redundant and Flexibility Coefficient

$$\theta_A + M_A \alpha_{AA} = 0$$

$$-\frac{PL^2}{16EI} + M_A\left(-\frac{L}{3EI}\right) = 0$$

Solve for M_A

$$M_A = \frac{PL^2}{16EI} \left(-\frac{3EI}{L}\right)$$

$$M_A = -\frac{3}{16}PL$$

Free Body Diagram



 $\frac{3}{16}PL$

Can now use equilibrium equations to find the remaining three unknowns

Find Remaining Unknowns



Can now use equilibrium equations to find the remaining three unknowns



Superposition of Primary and Redundant Problems

