## **Chapter 6. Manipulator Dynamics**

### 11-3-14

## Quiz on Nov. 11 on Homework #8

#### *Homework* #8. *Not collected*.

Solve 6.1 (Answer partially given in the textbook). 6.12 (Answer given). 6.16. Show how (6.32) is derived from (6.15) and (5.45).

Trace the steps taken to derive (6.36) from (6.12).

*Verify the formulation of (6.42).* 

See the Example in Section 6.7 - Two link robot arm with simplifying assumptions. Check the vector cross multiplications at several places in the solution.

#### Acceleration of Rigid Body – Definition:

Acceleration of linear velocity vector  $V_Q$  in frame  $\{B\}$ 

$${}^{B}\dot{V}s_{Q} = \frac{d}{dt}{}^{B}V_{Q} = \lim_{\Delta t \to 0} \frac{{}^{B}V_{Q}(t + \Delta t) - {}^{B}V_{Q}(t)}{\Delta t}$$
(6.1)

Acceleration of angular velocity vector  $\omega_{Q}$  in frame {B}

$${}^{A}\dot{\Omega}_{\varrho} = \frac{d}{dt}{}^{A}\Omega_{\varrho} = \lim_{\Delta t \to 0} \frac{{}^{A}\Omega_{\varrho}(t + \Delta t) - {}^{A}\Omega_{\varrho}(t)}{\Delta t}$$
(6.2)

#### Linear Acceleration:

From (5.12),

$${}^{A}V_{Q} = \frac{d}{dt} ({}^{A}_{B}R^{B}Q) = {}^{A}_{B}R^{B}V_{Q} + {}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}Q$$

$$(6.5)$$

Differentiating (6.5) and a term for linear acceleration of the origin of  $\{B\}$ ,

$${}^{A}\dot{V}_{Q} = \frac{d}{dt} ({}^{A}_{B}R^{B}V_{Q}) + {}^{A}\dot{\Omega}_{B} \times {}^{A}_{B}R^{B}Q + {}^{A}\Omega_{B} \times \frac{d}{dt} ({}^{A}_{B}R^{B}Q)$$
(6.7)

$$= ({}^{A}_{B}R^{B}\dot{V}_{Q} + {}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}V_{Q}) + {}^{A}\dot{\Omega}_{B} \times {}^{A}_{B}R^{B}Q + {}^{A}\Omega_{B} \times ({}^{A}_{B}R^{B}V_{Q} + {}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}Q)$$
(6.8)

With the linear acceleration of  $\{B\}_{Orig}$ 

$${}^{A}\dot{V}_{Q} = {}^{A}\dot{V}_{BO_{rg}} + {}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}V_{Q} + 2{}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}V_{Q} {}^{A}_{B}R^{B}\dot{V}_{Q} + {}^{A}\dot{\Omega}_{B} \times {}^{A}_{B}R^{B}Q + {}^{A}\Omega_{B} \times ({}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}Q)$$

When  ${}^{B}Q$  is constant,

$${}^{A}\dot{V}_{Q} = {}^{A}\dot{V}_{BO_{rg}} + {}^{A}_{B}R^{B}\dot{V}_{Q} + {}^{A}\dot{\Omega}_{B} \times {}^{A}_{B}R^{B}Q + {}^{A}\Omega_{B} \times ({}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}Q)$$
(6.12)

# Angular Acceleration:

To find the angular acceleration of {C} w.r.t. {A}, differentiate

$${}^{A}\Omega_{C} = {}^{A}\Omega_{B} + {}^{A}_{B}R^{B}\Omega_{C}$$

$$(6.13)$$

$${}^{A}\dot{\Omega}_{C} = {}^{A}\dot{\Omega}_{B} + \frac{d}{dt} ({}^{A}_{B}R^{B}\Omega_{C}) = {}^{A}\dot{\Omega}_{B} + {}^{A}_{B}R^{B}\dot{\Omega}_{C} + {}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}\Omega_{C}$$
(6.15)

# Rigid Body Mass Distribution

Inertia tensor – Describes the distribution of the mass around the center of a rigid body.



<sup>A</sup>P is the location vector of the differential volume dv.

Inertia Tensor of {A}: 
$${}^{A}I = \begin{bmatrix} I_{xx} & -I_{xy} & -I_{xz} \\ -I_{xy} & I_{yy} & -I_{yz} \\ -I_{xz} & -I_{yz} & I_{zz} \end{bmatrix}$$

Mass moment of inertia:

$$I_{xx} = \iiint_{V} (y^{2} + z^{2}) \rho d\upsilon \quad I_{yy} = \iiint_{V} (x^{2} + z^{2}) \rho d\upsilon \quad I_{zz} = \iiint_{V} (x^{2} + y^{2}) \rho d\upsilon$$
$$I_{xy} = \iiint_{V} xy \rho d\upsilon \qquad I_{xz} = \iiint_{V} xz \rho d\upsilon \qquad I_{yz} = \iiint_{V} yz \rho d\upsilon$$

Example 6.1

$$I_{xx} = \int_{0}^{h} \int_{0}^{l} \int_{0}^{w} (y^{2} + z^{2}) \rho dx dy dz = \frac{m}{3} (l^{2} + h^{2})$$

$$I_{xy} = \int_{0}^{h} \int_{0}^{l} xy \rho dx dy dz = \frac{m}{4} wl$$

$$H = \int_{0}^{h} \int_{0}^{l} y \rho dx dy dz = \frac{m}{4} wl$$

#### Parallel Axis Theorem:

Inertial tensor of a mass in frame  $\{A\}$  w.r.t. frame  $\{C\}$  with its origin at the center of the mass.

$${}^{A}I_{zz} = {}^{C}I_{zz} + m(x_{c}^{2} + y_{c}^{2})$$

$${}^{A}I_{xy} = {}^{C}I_{xy} - mx_{c}y_{c}$$

$${}^{A}P_{c} = \begin{bmatrix} x_{c} & y_{c} & z_{c} \end{bmatrix}^{T} - \text{Location of the center of mass in } \{A\}.$$
Example 6.2
The frame  $\{A\}$  has its origin at  ${}^{A}P_{c} = \frac{1}{2}\begin{bmatrix} w & l & h \end{bmatrix}^{T}$ 

$${}^{C}I_{zz} = \frac{m}{12}(w^{2} + l^{2}) \qquad {}^{C}I_{xy} = 0$$

*Newton's Equation on Force:*  $F = m\dot{v}_C$  at the center of mass *Euler's Equation on Moment:*  $N = {}^{C}I\dot{\omega} + \omega \times {}^{C}I\omega$  at the center of mass  ${}^{C}I =$  inertia tensor in frame {C} with its origin at the mass center

#### Newton-Euler Dynamic Equations

#### **Derivation of angular acceleration**

Forward angular velocity propagation

$${}^{i+1}\omega_{i+1} = {}^{i+1}_{i}R^{i}\omega_{i} + \dot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1}$$

$${}^{A}\dot{\Omega}_{C} = {}^{A}\dot{\Omega}_{B} + {}^{A}_{B}R^{B}\dot{\Omega}_{C} + {}^{A}\Omega_{B} \times {}^{A}_{B}R^{B}\Omega_{C}$$
(6.15)

#### Follow the derivation of (6.32) from (6.15) and (5.45)

Rewriting  $\{C\}$  with  $\{i+1\}$  and from (5.45)

$${}^{i+1}\dot{\omega}_{i+1} = {}^{i+1}_{i}R^{i}\dot{\omega}_{i} + {}^{i+1}_{i}R^{i}\omega_{i} \times \dot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1} + \ddot{\theta}_{i+1}{}^{i+1}\hat{Z}_{i+1}$$
(6.32)

For prismatic joints

$$\dot{\omega}_{i+1} = \dot{\omega}_{i+1} = \dot{R}^i \dot{\omega}_i$$

# **Derivation of linear acceleration**

From (6.12) and following similar steps taken for angular acceleration,

$${}^{i+1}\dot{\upsilon}_{i+1} = {}^{i+1}_{i}R[{}^{i}\dot{\omega}_{i} \times {}^{i}P_{i+1} + {}^{i}\omega_{i} \times ({}^{i}\omega_{i} \times {}^{i}P_{i+1}) + {}^{i}\dot{\upsilon}_{i}]$$
(6.34)

For *prismatic joints*, add two more terms to (6.34) per (6.10)

$${}^{i+1}\dot{\upsilon}_{i+1} = {}^{i+1}_{i}R[{}^{i}\dot{\omega}_{i} \times {}^{i}P_{i+1} + {}^{i}\omega_{i} \times ({}^{i}\omega_{i} \times {}^{i}P_{i+1}) + {}^{i}\dot{\upsilon}_{i}] + 2{}^{i+1}\omega_{i+1} \times \dot{d}_{i+1} \overset{i+1}{\sim} \hat{Z}_{i+1} + \ddot{d}_{i+1} \overset{i+1}{\sim} \hat{Z}_{i+1} \quad (6.35)$$

*Linear acceleration of the center of mass*, from (6.12)

# Trace the steps taken in applying (6.12),

$${}^{i}\dot{\upsilon}_{Ci} = {}^{i}\dot{\omega}_{i} \times {}^{i}P_{Ci} + {}^{i}\omega_{i} \times ({}^{i}\omega_{i} \times {}^{i}P_{Ci}) + {}^{i}\dot{v}_{i}$$

$$(6.36)$$

## The inertial force and torque acting at the center of the mass:

From (6.32) and (6.36)

$$F_{i} = m\dot{\upsilon}_{c_{i}}$$

$$N_{i} = {}^{C_{i}}I\dot{\omega}_{i} + \omega_{i} \times {}^{C_{i}}I\omega_{i}$$
(6.37)



Force and torque balance equations at the center of mass of link i:

$${}^{i}F_{i} = {}^{i}f_{i} - {}^{i}_{i+1}R^{i+1}f_{i+1}$$
(6.38)

$${}^{i}N_{i} = {}^{i}n_{i} - {}^{i}n_{i+1} + ({}^{i}P_{i} - {}^{i}P_{i}) \times {}^{i}f_{i} - ({}^{i}P_{i+1} - {}^{i}P_{i}) \times {}^{i}f_{i+1}$$

$${}^{i}P_{i} = 0$$
(6.39)

Rearranging the equations and adding rotations;

$${}^{i}f_{i} = {}^{i}_{i+1}R^{i+1}f_{i+1} + {}^{i}F_{i}$$
(6.41)

#### Figure out how this equation is related to (6.39).

$${}^{i}n_{i} = {}^{i}N_{i} + {}^{i}_{i+1}R^{i+1}n_{i+1} + {}^{i}P_{Ci} \times {}^{i}F_{i} + {}^{i}P_{i+1} \times {}^{i}_{i+1}R^{i+1}f_{i+1}$$
(6.42)

Finally, the joint torque is the Z component of the vector representing the inertial torque:

$$\tau_i = n_i^{T_i} \hat{Z}_i \tag{6.43}$$

For prismatic joints:

$$\tau_i = f_i^{T_i} \hat{Z}_i \tag{6.44}$$

Forward and backward iterations: Eq (6.45)-(6.53)

Forward - Link velocities and accelerations via the Newton-Euler (6.31)-(6.37). Backward - Find joint forces and torques via (6.38)-(6.44).

# See the Example in Section 6.7 – Simplified two link robot arm. Check the vector cross multiplications at several places in the solution.

#### **Dynamic Equations**

State Space equation:

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta,\dot{\Theta}) + G(\Theta)$$
(6.59)

...

where,

$$M(\Theta) = n \ge n \mod n$$
 mass matrix of terms containing  $\theta_i, i = 1..n$ 

 $V(\Theta, \dot{\Theta}) = n \ge 1$  vector of centrifugal and Coriolis terms containing  $\dot{\theta}_i, i = 1..n$  $G(\Theta) = n \ge 1$  vector containing a "g" gravity term.

Configuration Space equation:

$$\tau = M(\Theta)\ddot{\Theta} + B(\Theta)\left[\dot{\Theta}\dot{\Theta}\right] + C(\Theta)\left[\dot{\Theta}^2\right] + G(\Theta)$$
(6.63)

where,

 $B_{r}(\Theta)$  = matrix of Coriolis coefficients

$$\begin{bmatrix} \dot{\Theta}\dot{\Theta} \end{bmatrix} = \begin{bmatrix} \dot{\theta}_{1}\dot{\theta}_{2} & \dot{\theta}_{1}\dot{\theta}_{3} & \dots & \dot{\theta}_{n-1}\dot{\theta}_{n} \end{bmatrix}, \text{ vector of joint velocity products}$$
(6.64)  
$$\begin{bmatrix} \dot{\Theta}^{2} \end{bmatrix} = \begin{bmatrix} \dot{\theta}_{1}^{2} & \dot{\theta}_{1}^{2} & \dots & \dot{\theta}_{n}^{2} \end{bmatrix}, \text{ matrix of centrifugal coefficients}$$
(6.65)

## Lagrangian Dynamic Formulationu

Quadratic form of manipulator kinetic energy, analogous to  $k = \frac{1}{2} mv^2$ 

$$k(\Theta, \ddot{\Theta}) = \frac{1}{2} \dot{\Theta}^{T} M(\Theta) \dot{\Theta}$$
(6.71)

(6.73)

Potential energy:  $u_i = -m_i^{0}gT^{0}P_{Ci} + u_{ref}$ 

$$u = \sum u_i$$

Lagrangian The difference between the kinetic energy and the potential energy

$$L(\Theta, \dot{\Theta}) = k(\Theta, \dot{\Theta}) - u(\Theta)$$
(6.75)

$$\tau = M(\Theta)\ddot{\Theta} + V(\Theta, \dot{\Theta}) + G(\Theta)$$
(6.59)

For Cartesian space,

$$F = M_{x}(\Theta)\ddot{X} + V_{x}(\Theta, \dot{\Theta}) + G_{x}(\Theta)$$
(6.91)

From

$$\tau = J^{T}(\Theta)F \rightarrow F = J^{-T}(\Theta)\tau$$

$$F = J^{-T}[M_{x}(\Theta)\ddot{\Theta} + V_{x}(\Theta,\dot{\Theta}) + G_{x}(\Theta)]$$
(6.94)

From

$$\dot{X} = J\dot{\Theta} \qquad \Rightarrow \ddot{X} = \dot{J}\dot{\Theta} + J\ddot{\Theta} \qquad \Rightarrow \ddot{\Theta} = J^{-1}\ddot{X} - J^{-1}\dot{J}\dot{\Theta} \qquad (6.97)$$

Substituting (6.97) into (6.94)

$$F = J^{-T} [M_x(\Theta)(J^{-1}\ddot{X} - J^{-1}\dot{J}\dot{\Theta}) + V_x(\Theta, \dot{\Theta}) + G_x(\Theta)]$$
(6.98)

Then,

$$M_{x}(\Theta) = J^{-T}M(\Theta)J^{-1}$$

$$V_{x}(\Theta, \dot{\Theta}) = J^{-T}[V(\Theta, \dot{\Theta}) - M(\Theta)J^{-1}\dot{J}\dot{\Theta}]$$

$$G_{x}(\Theta) = J^{-T}G(\Theta)$$
(6.99)

Cartesian configuration space torque

$$\tau = J^{T}[M_{x}(\Theta)\ddot{X} + V_{x}(\Theta, \dot{\Theta}) + G_{x}(\Theta)]$$
(6.104)

$$\tau = J^{T} M(\Theta) \ddot{X} + B_{x}(\Theta) \left[ \dot{\Theta} \dot{\Theta} \right] + C_{x}(\Theta) \left[ \dot{\Theta}^{2} \right] + G_{x}(\Theta)$$
(6.105)

where,

 $B_x(\Theta)$  = matrix of Coriolis coefficients

$$C_{x}(\Theta) = \text{matrix of centrifugal coefficients}$$
$$\begin{bmatrix} \dot{\Theta}\dot{\Theta} \end{bmatrix} = \begin{bmatrix} \dot{\theta}_{1}\dot{\theta}_{2} & \dot{\theta}_{1}\dot{\theta}_{3} & \dots & \dot{\theta}_{n-1}\dot{\theta}_{n} \end{bmatrix}, \text{ a vector of the joint velocity products}$$
$$\begin{bmatrix} \dot{\Theta}^{2} \end{bmatrix} = \begin{bmatrix} \dot{\theta}_{1}^{2} & \dot{\theta}_{1}^{2} & \dots & \dot{\theta}_{n}^{2} \end{bmatrix}, \text{ a matrix of the centrifugal coefficients}$$

# Friction

Friction force  $F(\Theta, \dot{\Theta})$  may be added to (6.59) or (6.104) to account for the effect of friction on *Simulation* 

Numerical integration method is used to solve the acceleration problem of the manipulator.

$$\ddot{\Theta} = M^{-1}(\Theta) \Big[ \tau - V \Big( \Theta, \dot{\Theta} \Big) - G \Big( \Theta \Big) - F \Big( \Theta, \dot{\Theta} \Big) \Big]$$

$$\dot{\Theta}(t + \Delta t) = \dot{\Theta}(t) + \ddot{\Theta}(\Delta t), \text{ and}$$

$$\Theta(t + \Delta t) = \Theta(t) + \dot{\Theta}(t) \Delta t + \frac{1}{2} \ddot{\Theta}(\Delta t) \Delta t^{2}$$
(6.117)